



Third Integer Resonances



CHESS & LEPP

Sextupole: $\Delta f = -k_2 \frac{1}{2} x^2$

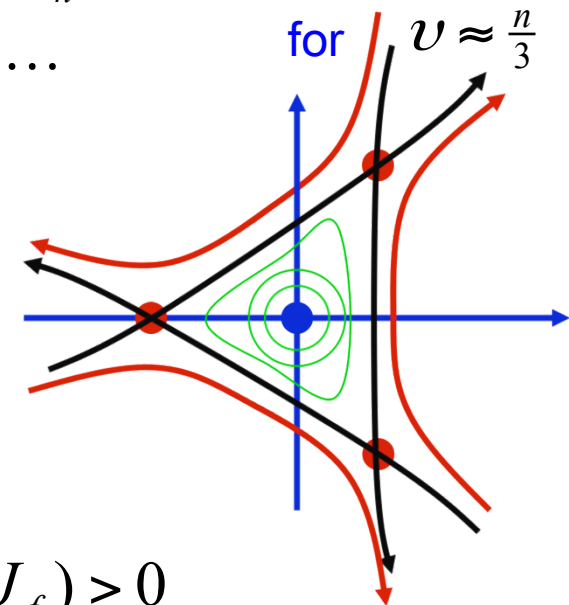
$$\begin{aligned}\Delta H &= \frac{L}{2\pi} k_2 \frac{1}{3!} x^3 = \frac{L}{2\pi} k_2 \frac{1}{3!} \sqrt{2J\beta}^3 \sin^3(\tilde{\psi} + \varphi) \\ &= \frac{L}{2\pi} k_2 \frac{1}{3!4} \sqrt{2J\beta}^3 [\sin(3[\tilde{\psi} + \varphi]) + 3\sin(\tilde{\psi} + \varphi)]\end{aligned}$$

Simplification: one sextupole $k_2(\vartheta) = k_2 \delta(\vartheta) = k_2 \frac{1}{2\pi} \sum_{n=-\infty}^{\infty} \cos(n\vartheta)$

$$\Delta H = \frac{L}{2\pi} k_2 \frac{1}{3!4} \sqrt{2J\beta}^3 \frac{1}{2\pi} \cos(-n\vartheta + 3\varphi + 3\tilde{\psi} - \frac{\pi}{2}) + \dots$$

$$\Delta H \approx A_2 \sqrt{J}^3 \cos(3\Phi)$$

$$\left. \begin{aligned} \Phi_f &= 0, \frac{1}{3}\pi, \frac{2}{3}\pi, \dots \\ \delta \pm A_2 \frac{3}{2} \sqrt{J} &= 0 \end{aligned} \right\} \Phi_f = \frac{1}{3}\pi, \pi, \frac{5}{3}\pi \quad \text{for } \delta > 0$$



All these fixed points are instable since $H_{nm}(J_f) H''_{nm}(J_f) > 0$



Fourth Integer Resonances



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Octupole: $\Delta f = -k_3 \frac{1}{3!} x^3$, $\Delta H = \frac{L}{2\pi} k_3 \frac{1}{4!} x^4 = \frac{L}{2\pi} k_3 \frac{1}{3!} J^2 \beta^2 \sin^4(\tilde{\psi} + \varphi)$

$$= \frac{L}{2\pi} k_3 \frac{1}{3!8} J^2 \beta^2 [\cos(4[\tilde{\psi} + \varphi]) - 4\cos(2[\tilde{\psi} + \varphi]) + 3]$$

Simplification: one octupole $k_3(\vartheta) = k_3 \delta(\vartheta) = k_3 \frac{1}{2\pi} \sum_{n=-\infty}^{\infty} \cos(n\vartheta)$

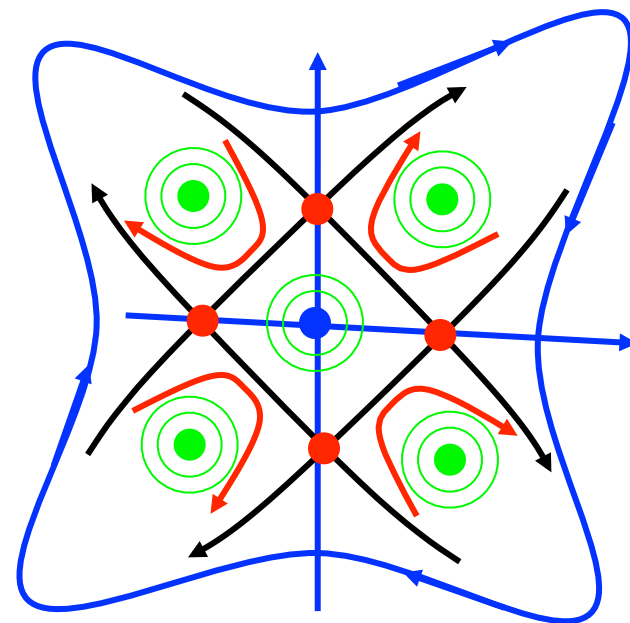
$$\Delta H \approx A_3 J^2 [3 + \cos(4\Phi)] \quad \text{for } \nu \approx \frac{n}{4}$$

$$\Phi_f = 0, \frac{1}{4}\pi, \frac{2}{4}\pi, \dots \quad \text{Either 8 fixed points: } \delta < 0$$

$$\delta + A_3 2J (3 \pm 1) = 0 \quad \text{or none for: } \delta > 0$$

$$H_{nm}(J_f)[H_{nm}''(J_f) \pm \Delta \nu'(J_f)] < 0$$

Stability for $(2A_3 J)^2 [1 \pm 3] < 0$,
i.e. for the 4 outer fixed points.





Resonance Width (Strength)

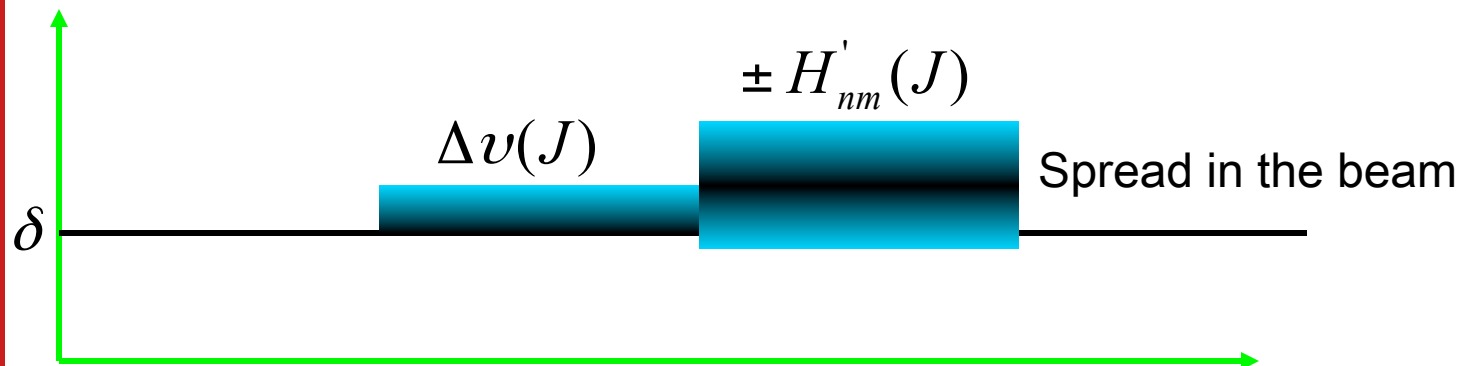


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Fixed points: $\frac{d}{d\vartheta} J = mH_{nm}(J_f)\sin(m\Phi_f) = 0 \Rightarrow \Phi_f = \frac{k}{m}\pi$

If $\delta + \Delta\nu(J_f) \pm H'_{nm}(J_f) = 0$ has a solution.

δ has to avoid the region $\delta + \Delta\nu(J) \pm H'_{nm}(J) = 0$ for all particles.



Assuming that the tune shift and perturbation are monotonous in J :

This tune region has the width $\Delta_{nm} = 2 |H'_{nm}(J_{\max})|$ for strong resonances.

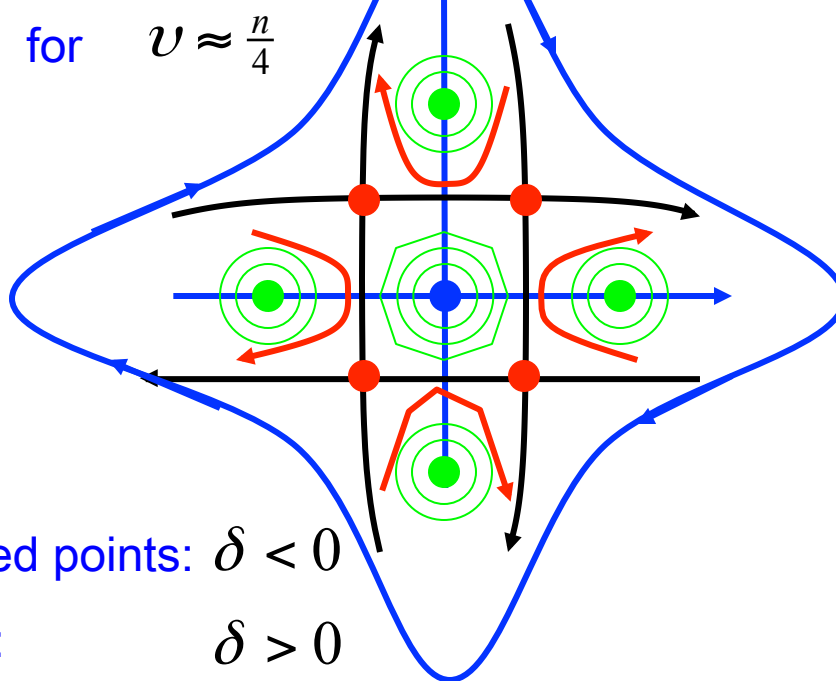
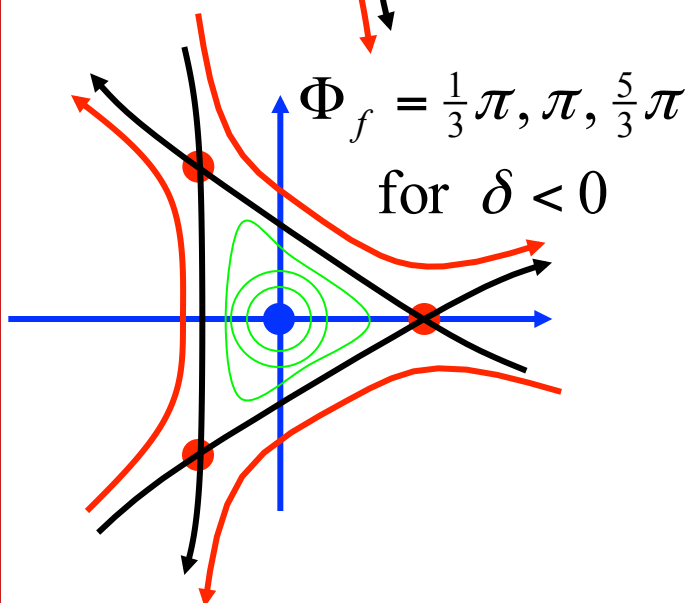
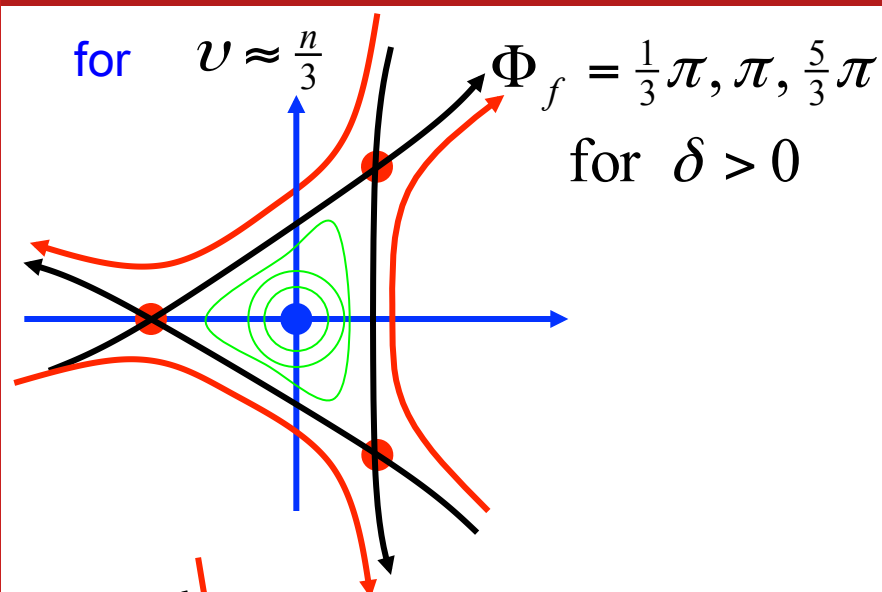
Δ_{nm} Is called **Resonance Width**, Resonance Strength, or Stop-Band Width



Particle motion in the single resonance model



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Either 8 fixed points: $\delta < 0$
or none for: $\delta > 0$

How can the motion inside the fixed points be simplified for a real accelerator ?

→ Normal Form Theory



$$\frac{d}{d\vartheta} J_x = \cos(\tilde{\psi}_x + \varphi_x) \sqrt{2J_x \beta_x} \Delta f_x \frac{L}{2\pi} \quad , \quad \frac{d}{d\vartheta} \varphi_x = \nu_x - \sin(\tilde{\psi}_x + \varphi_x) \sqrt{\frac{\beta_x}{2J_x}} \Delta f_x \frac{L}{2\pi}$$

$$\frac{d}{d\vartheta} J_y = \cos(\tilde{\psi}_y + \varphi_y) \sqrt{2J_y \beta_y} \Delta f_y \frac{L}{2\pi} \quad , \quad \frac{d}{d\vartheta} \varphi_y = \nu_y - \sin(\tilde{\psi}_y + \varphi_y) \sqrt{\frac{\beta_y}{2J_y}} \Delta f_y \frac{L}{2\pi}$$

$$\frac{d}{d\vartheta} \vec{\varphi} = \vec{\partial}_J H \quad , \quad \frac{d}{d\vartheta} \vec{J} = -\vec{\partial}_{\varphi} H \quad , \quad H(\vec{\varphi}, \vec{J}, \vartheta) = \vec{\nu} \cdot \vec{J} - \frac{L}{2\pi} \int_0^{\vec{x}} \Delta \vec{f}(\hat{x}, s) d\hat{x}$$

The integral form can be chosen since it is path independent. This is due to the Hamiltonian nature of the force:

$$\Delta f_{x,y}(x, y, s) = -\partial_{x,y} \Delta H(x, y, s)$$

Single Resonance model for two dimensions means retaining only the amplitude dependent tune shift and one term in the two dimensional Fourier expansion:

$$H(\vec{\varphi}, \vec{J}, \vartheta) = \vec{\nu} \cdot \vec{J} + H_{00}(\vec{J}) + H_{n\vec{m}}(\vec{J}) \cos(n\vartheta + m_x \varphi_x + m_y \varphi_y + \Psi_{n\vec{m}}(\vec{J}))$$

For $n + m_x \nu_x + m_y \nu_y \approx 0$

$$m_x \varphi_x + m_y \varphi_y = \vec{m} \cdot \vec{\varphi}$$



Sum and Difference Resonances



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$n + m_x \nu_x + m_y \nu_y \approx 0$ means that oscillations in y can drive oscillations in x in

$$x'' = -K x + \Delta f_x(x, y, s)$$

The resonance term in the Hamiltonian then changes only slowly:

$$H(\vec{\varphi}, \vec{J}, \vartheta) = \vec{\nu} \cdot \vec{J} + H_{00}(\vec{J}) + H_{n\vec{m}}(\vec{J}) \cos(n\vartheta + \vec{m} \cdot \vec{\varphi} + \Psi_{n\vec{m}}(\vec{J}))$$

$$\frac{d}{d\vartheta} \vec{\varphi} = \vec{\partial}_J H, \quad \frac{d}{d\vartheta} \vec{J} = -\vec{\partial}_\varphi H$$

$$J = \vec{m} \cdot \vec{J}$$

$$J_\perp = m_x J_x - m_y J_y = \vec{m} \times \vec{J} \Rightarrow \frac{d}{d\vartheta} J_\perp = 0$$

Difference resonances lead to stable motion since:

$$n + |m_x| \nu_x - |m_y| \nu_y \approx 0 \Rightarrow |m_x| J_x + |m_y| J_y = \text{const.}$$

Sum resonances lead to unstable motion since:

$$n + |m_x| \nu_x + |m_y| \nu_y \approx 0 \Rightarrow |m_x| J_x - |m_y| J_y = \text{const.}$$



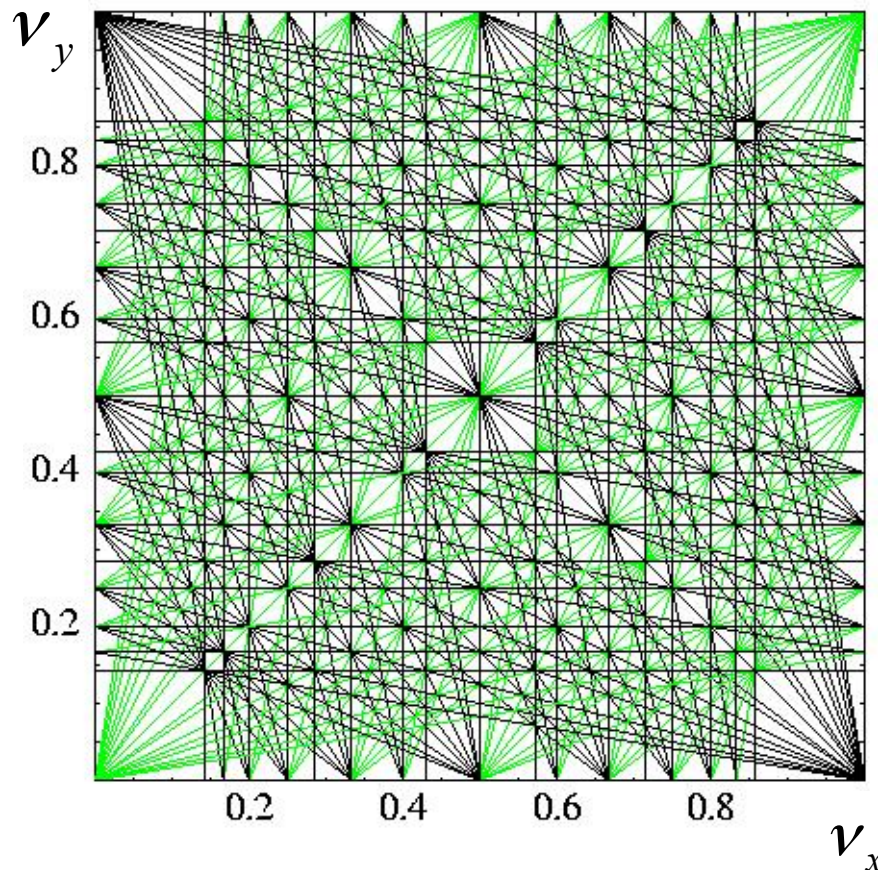
Resonances Diagram



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$n + m_x \nu_x + m_y \nu_y \approx 0$ means that oscillations in y can drive oscillations in x in

$$x'' = -K x + \Delta f_x(x, y, s)$$



All these resonances have to be avoided by their respective resonance width.

The position of an accelerator in the tune plane is called its Working Point.